

# APPARATUS AND METHOD FOR CONTROLLING LINEAR COMPRESSOR

## BACKGROUND OF THE INVENTION

### Field of the Invention

5 The present invention relates generally to linear compressors, and more particularly to an apparatus and method for controlling a linear compressor, which prevents the collision of a piston with a valve to improve operational efficiency of the linear compressor during operation of the  
10 linear compressor.

### Description of the Prior Art

Fig. 1 is block diagram of a conventional linear compressor control apparatus.

15 Referring to Fig. 1, the conventional linear compressor control apparatus comprises a core 10, first and second coils 12 and 13, and a signal processing unit 20. The core 10 of a magnetic substance operates in conjunction with a machine for detecting a position of a piston. The first and second coils 12 and 13 are symmetrically wound around the outside of the  
20 core 10. The signal processing unit 20 detects and outputs the change of the core position according to voltages induced in the first and second coils 12 and 13.

The signal processing unit 20 comprises a first full-wave rectifying unit 21, a second full-wave rectifying unit 22, a differential amplifying  
25 unit 23, a filter unit 24 and a peak detection unit 25. The first full-wave

10  
15  
20  
25

rectifying unit 21 full-wave rectifies the voltage induced in the first coil 12, and the second full-wave rectifying unit 22 full-wave rectifies the voltage induced in the second coil 13. The differential amplifying unit 23 amplifies the voltage difference between the rectified voltages of the first and second full-wave rectifying units 21 and 22. The filter unit 24 removes high frequency component of an output signal from the differential amplifying unit 23. The peak detection unit 25 detects the maximum value and the minimum value of an output signal from the filter unit 24, and transmits the detected values to a control unit.

The operation of the conventional apparatus having the above construction is described.

When an AC power voltage of several KHz is applied to both the first and second coils 12 and 13 from the outside, if the position of the core 10 is changed due to the change of position of the machine for detecting the position of the piston, voltages proportional to the change in position of the core 10 are induced in the first and second coils 12 and 13. The voltages induced in the first and second coils 12 and 13 are full-wave rectified by the first and second full-wave rectifying units 21 and 22, respectively, and the rectified results are applied to input terminals of the differential amplifying unit 23.

The differential amplifying unit 23 amplifies the voltage difference between the full-wave rectified voltages of the first and second full-wave rectifying units 21 and 22, and outputs the amplified results to the filter unit 24. Then, the filter unit 24 removes the high frequency component of the output signal from the differential amplifying unit 23, amplifies the

resulting signal, and outputs the amplified signal to the peak detection unit 25. The peak detection unit 25 full-wave rectifies the output signal from the filter unit 24 and outputs the rectified signal to the microcontroller 30. The microcontroller 30 controls the stroke of the linear compressor in response to the output signal from the peak detection unit 30, which is obtained by full-wave rectifying the output signal from the filter unit 24.

The conventional linear compressor control apparatus has a constant stroke by controlling only the stroke of the piston of the linear compressor according to the above construction. However, the conventional linear compressor control apparatus is disadvantageous in that it cannot maintain a constant top clearance with respect to the position of its top dead center due to a characteristic of the linear compressor that the center position of the piston is changed according to a load.

#### SUMMARY OF THE INVENTION

Accordingly, the present invention has been made keeping in mind the above problems occurring in the prior art, and an object of the present invention is to provide an apparatus and method for controlling a linear compressor, which prevents the collision of a piston of the linear compressor with a valve to improve operational efficiency of the linear compressor by controlling a top clearance with respect to the top dead center of the piston.

In accordance with one aspect of the present invention, the above and

other objects can be accomplished by the provision an apparatus for controlling a linear compressor, comprising a collision detection unit for detecting a collision of a piston with a valve due to the operations of the linear compressor; a control unit for determining whether the collision of the piston occurs on the basis of an output signal from the collision detection unit, and resetting maximum amplitude data of the piston of the linear compressor when the collision occurs; and a compressor driving unit for controlling the maximum amplitude of the piston of the linear compressor under the control of the control unit.

In accordance with another aspect of the present invention, there is provided a method for controlling a linear compressor, comprising the steps of a) presetting a maximum amplitude of a piston of the linear compressor; b) detecting a signal when the linear compressor operates; c) determining whether any collision of the piston has occurred on the basis of the detected signal; d) resetting the maximum amplitude if it is determined that a collision of the piston has occurred at step c); and e) driving the linear compressor according to the reset maximum amplitude.

#### BRIEF DESCRIPTION OF THE DRAWINGS

The above and other objects, features and other advantages of the present invention will be more clearly understood from the following detailed description taken in conjunction with the accompanying drawings, in which:

Fig. 1 is a block diagram of a conventional linear compressor control

apparatus;

Fig. 2 is a block diagram of a linear compressor control apparatus according to a preferred embodiment of the present invention;

Fig. 3 is a detailed circuit diagram of a collision detection unit  
5 included in the apparatus of this invention;

Fig. 4 is a flowchart of a linear compressor control method of this invention; and

Fig. 5 is a graphic view showing the variation of dynamic characteristics according to the collision of the piston of this invention.

10  
DESCRIPTION OF THE PREFERRED EMBODIMENTS

Fig. 2 is a block diagram of a linear compressor control apparatus according to a preferred embodiment of this invention.

15 Referring to Fig. 2, the linear compressor control apparatus comprises a control unit 330, a compressor driving unit 350, a collision detection unit 200, an amplitude calculation unit 310, and a displacement calculation unit 320. The control unit 330 controls the overall operation of the linear compressor control apparatus, and the compressor driving unit  
20 350 controls the operation of a linear compressor 100 under the control of the control unit 330. The collision detection unit 200 detects the collision of a piston according to the operation of the linear compressor 100. The amplitude calculation unit 310 calculates the amplitude of the piston on the basis of an output signal from the collision detection unit  
25 200, and the displacement calculation unit 320 calculates the displacement

of the piston. Further, the linear compressor control apparatus comprises a first storage unit 341 for storing preset maximum amplitude data, and a second storage unit 342 for storing reset maximum amplitude data.

Fig. 3 is a detailed circuit diagram of the collision detection unit  
5 200 of this invention.

Referring to Fig. 3, the collision detection unit 200 includes a  
bridge unit 220, a core 221, a sine wave generating unit 210, first and  
second half-wave rectifying units 231 and 232, a differential amplifying  
unit 240, a low pass filter 250, and a peak detection unit 260. The bridge  
10 unit 220 has first and second coils L1 and L2 serially connected to the  
ground, and resistors R1 and R2 connected in parallel with the coils L1 and  
L2 and serially connected to each other. The core 221 of a magnetic  
substance linearly reciprocates while penetrating the wound coils L1 and L2  
according to the movement of the piston of the linear compressor 100. The  
15 sine wave generating unit 210 generates a sine wave of several KHz and  
provides the sine wave to the first and second coils L1 and L2. The first  
and second half-wave rectifying units 231 and 232, each comprised of a  
diode, half-wave rectify an output signal A from the junction of the  
resistors R1 and R2, and an output signal B from the junction of the first  
20 and second coils L1 and L2, respectively. The differential amplifier 240  
differentially amplifies output signals from the first and second half-wave  
rectifying units 231 and 232. The low pass filter 250 is used for low-pass  
filtering an output signal from the differential amplifying unit 240. The  
peak detection unit 260 detects the peak of an output signal from the low  
25 pass filter 250, and outputs the detected result to the control unit 330.

The differential amplifying unit 240 has an operational amplifier IC1 in which a resistor R3 and a resistor R4 are serially connected to the non-inverting and inverting input terminals thereof, respectively. Further, a resistor R5 is connected between the inverting input terminal of the amplifier IC1 and the ground, and a resistor R6 is connected between the non-inverting input terminal and the output terminal of the amplifier IC1.

*Inst a1*

~~The low pass filter 250 has an operation amplifier IC2 whose non-inverting input terminal is connected to an output terminal of the differential amplifying unit 240 through the resistor R6, and the inverting input terminal is connected to the ground. Further, a resistor R8 and a capacitor C1 are connected in parallel with each other between the non-inverting input terminal and the output terminal of the operational amplifier IC2.~~

TO THE

*Inst a2*

~~The peak detection unit 260 detects a unidirectional movement of the piston so as to minimize the circuit size, and is provided with a diode D3, a resistor R9, a capacitor C2, and a resistor R10. The diode D3 is connected to the output terminal of the operational amplifier IC2 of the low pass filter 250 to half-wave rectify the output signal from the operation amplifier IC2. The resistor R9 is serially connected between an output terminal of the diode D3 and the control unit 330. The capacitor C2 is connected between the output terminal of the peak detection unit 260 and the ground so as to smooth the output signal from the peak detection unit 260. The resistor R10 is connected between the output terminal of the diode D3 and the ground.~~

Hereinafter, the control method of this invention is described in

detail.

Fig. 4 is a flowchart of a linear compressor controlling method of this invention.

Referring to Fig. 4, the control unit 330 loads the data stored in the first storage unit 341, and sets a maximum amplitude of the piston of the linear compressor 100 at step S10. The maximum amplitude is the maximum value for allowing the piston of the linear compressor 100 to reciprocate without any collision, is preset when the linear compressor 100 is manufactured, and is stored in the first storage unit 341.

After setting the maximum amplitude, the control unit 330 controls the compressor driving unit 350 to operate the linear compressor 100 using a typical operating method at step S20. When the linear compressor 100 operates, the control unit 330 detects a signal through the collision detection unit 200 at step S30.

The operation of the collision detection unit 200 is described as follows.

The sine wave of several KHz from the sine wave generating unit 210 is provided to the resistors R1 and R2, and the first and second coils L1 and L2 of the bridge unit 220.

When the core 221 made of a magnetic substance linearly reciprocates according to the operation of the piston (not shown) of the linear compressor 100, a magnetic field is changed as much as the position of the core 221 is changed. Accordingly, voltages proportional to the change in position of the core 221 are induced in the first and second coils L1 and L2.



The voltages induced in the first and second coils L1 and L2 are full-wave rectified by a diode D1 of the first half-wave rectifying unit 231 and a diode D2 of the second half-wave rectifying unit 232, respectively, and the rectified voltages are transmitted to the differential amplifying unit 240.

The output signal from the diode D1 is applied to the non-inverting terminal of the operational amplifier IC1 through the resistor R3, while the output signal from the diode D2 is applied to the inverting terminal of the operational amplifier IC1 through the resistor R4. Thereby, the operational amplifier IC1 differentially amplifies the input signals applied to the non-inverting and inverting input terminals thereof.

The output signal from the differential amplifying unit 240 is applied to both the low pass filter 250 and the amplitude calculation unit 310. The low pass filter 250 removes high frequency noise component generated by the sine wave generating unit 210 from the output signal of the differential amplifying unit 240, and outputs the noise-removed signal to the peak detection unit 260. The peak detection unit 260 detects the peak of the input signal applied thereto and outputs the detected result to the control unit 330.

Further, the amplitude calculation unit 310 calculates the amplitude of the piston and outputs the calculated amplitude to the control unit 330. The displacement calculation unit 320 calculates the displacement of the piston on the basis of the amplitude data calculated by the amplitude calculation unit 310, and outputs the calculated displacement to the control unit 330.

Accordingly, the control unit 330 can detect both whether the collision of the piston with a valve occurs and the amplitude and displacement of the piston, on the basis of the output signals from the peak detection unit 260, the amplitude calculation unit 310 and the displacement calculation unit 320.

As described above, after signal detection at step S30, the control unit 330 determines whether the collision of the piston with a valve has occurred at step S40. At step S40, if it is determined that the collision has occurred, the control unit 330 resets the maximum amplitude at step S41. In this case, the <sup>of the piston</sup> (maximum amplitude) is reset by subtracting the preset maximum <sup>50</sup> amplitude value from the <sup>60</sup> amplitude value obtained when the collision occurs. The control unit 330 stores the reset maximum amplitude data in the second storage unit 342.

After resetting the maximum amplitude at step S41, the control unit 330 determines whether the linear compressor 100 should be stopped in response to an external signal at step S50. If it is determined that linear compressor 100 should not be stopped in response to the external signal at step S50, the control unit 330 controls the operation of the linear compressor 100 through the compressor driving unit 350, depending on the reset maximum amplitude data at step S20. ←

On the other hand, if it is determined that the linear compressor 100 should be stopped in response to the external signal, the control unit 330 stops the operation of the linear compressor 100 through the compressor driving unit 350.

Fig. 5 is a graphic view showing the variation of the dynamic

characteristics due to the collision of the piston in accordance with this invention. Referring to Fig. 5, A is the top dead center of the piston when the collision occurs, and B is the top dead center of the re-controlled piston after the collision occurs. Fig. 5 shows that the collision can be prevented by resetting the top dead center of the piston when the collision of the piston occurs during an operation of the linear compressor 100.

As described above, the present invention provides an apparatus and method for controlling a linear compressor, which minimizes collision of a piston of the linear compressor with a valve by minimizing the top clearance of the linear compressor, thus enabling the linear compressor to maintain a high efficient operation. Further, the present invention is advantageous in that it determines only a unidirectional moving distance, thereby minimizing the entire circuit size.

Although the preferred embodiments of the present invention have been disclosed for illustrative purposes, those skilled in the art will appreciate that various modifications, additions and substitutions are possible, without departing from the scope and spirit of the invention as disclosed in the accompanying claims.